

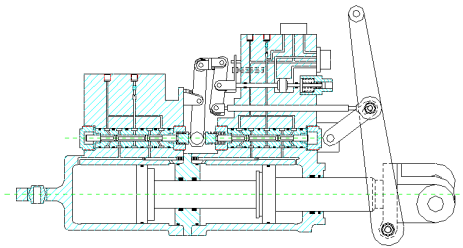
THE EXPERIMENTAL DYNAMIC IDENTIFICATION OF THE HYDRAULIC SERVOMECHANISMS

MATHEMATICAL MODEL:

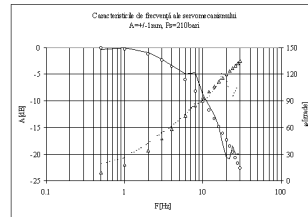
| | | | |
|---------------------|---|--|--|
| TRANSFER FUNCTION | $G(s) = \frac{K}{1+s \cdot T} e^{-s\tau}$ | $G(s) = \frac{K \cdot \omega_0^2}{s^2 + 2\xi \cdot \omega_0 \cdot s + \omega_0^2}$ | |
| MOTION EQUATION | $T \frac{dy(t)}{dt} + y(t) = K \cdot u(t - \tau)$ | $\frac{d^2y(t)}{dt^2} + 2 \cdot \xi \cdot \omega_0 \cdot \frac{dy(t)}{dt} + \omega_0^2 \cdot y(t) = K \cdot \omega_0^2 \cdot u(t)$ | |
| ANALITICAL SOLUTION | $y(t) = K \left(1 - e^{-\frac{t-\tau}{T}} \right) \cdot u(t - \tau)$ | $y(t) = K \cdot \left[1 - \frac{1}{\sqrt{1-\xi^2}} e^{-\xi \cdot \omega_0 \cdot t} \cdot \sin(\omega_0 \cdot \sqrt{1-\xi^2} \cdot t + \varphi) \right]$ | |

EXPERIMENTAL RESULTS

$$\xi = \frac{-\ln M}{\sqrt{\pi^2 + (\ln M)^2}} \quad \omega_0 = \frac{2 \cdot \pi}{T \cdot \sqrt{1-\xi^2}} \quad \varphi = \arccos \xi$$



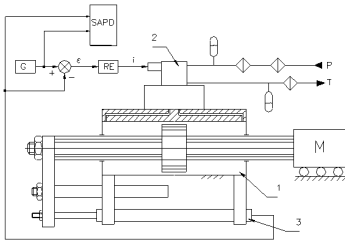
Hydraulic servo mechanism



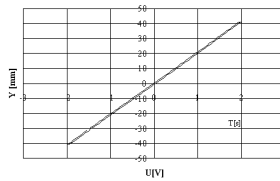
Frequency characteristics of a hydraulic servo mechanism;
(- measured values, o-values of identification model)

The transfer function of the identified servo mechanism

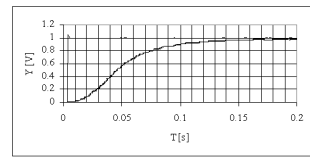
$$H(s) = \frac{3428}{s^2 + 170,9 \cdot s + 3403}$$



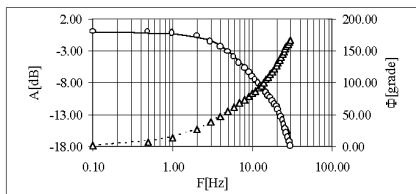
Electrohydraulic servo mechanism



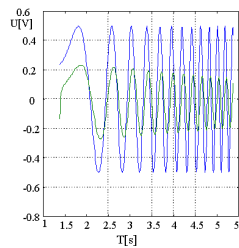
Steady state characteristic of electrohydraulic servo mechanism



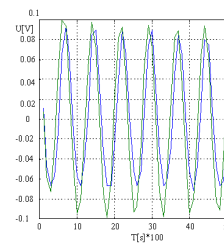
Step input response of the electrohydraulic servo mechanism; $T=0.055$



Frequency characteristics of the electrohydraulic servo mechanism;
(- measured values; o-values of identification model)



Input and output signals of the servomechanism
(linear variable frequency)



The servomechanism and ARX model output

Identified transfer function of the electrohydraulic model

$$H(s) = \frac{868100}{s^2 + 3s + 2}$$

